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Thermal Characteristics of Externally Venting Flames and their Effect on the Exposed Façade Surface

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ABSTRACT

In a compartment fire, Externally Venting Flames (EVF) may significantly increase the risk of fire spreading to adjacent floors or buildings, especially when combustible insulation materials are installed on the building façade. An increasing number of recent reports suggest that existing fire engineering design methodologies cannot describe with sufficient accuracy the characteristics of EVF under realistic fire load conditions. In this context, a series of fire safety engineering design correlations used to describe the main EVF thermal characteristics, namely EVF centreline temperature and EVF-induced heat flux on the exposed façade surface, are comparatively assessed. Towards this end, measurements obtained in a medium- and a large-scale compartment-façade fire test are employed; aiming to broaden the scope of the validation study, predictions of the investigated correlations are further compared to measurements obtained in 6 large-scale fire tests found in the literature. It is found that the correlation proposed in EN1991-1-2 (Eurocode 1) for the estimation of the EVF centreline temperature is under-predicting the measured values in large-scale fire tests. In addition, it is concluded that estimation of the local flame emissivity should take into account the specific fuel type used in each case.

KEYWORDS: compartment fires; heat transfer; externally venting flames; façade fires; Eurocode.

NOMENCLATURE LISTING

A_o	Opening area (m ²)	T_z	EVF centreline temperature, related to the height from the opening lintel (K)
A_v	Area of vertical openings (m ²)	T_{wall}	Wall temperature (K)
c	Empirical factor (valued 4.67)	t_{dur}	Total fire duration (s)
C_p	Specific heat capacity (J/kg K)	U	External wind speed (m/s)
d_{eq}	Characteristic length scale of an external structural element (m)	w_f	EVF width (m)
g	Gravitational acceleration (9.81 m/s ²)	W_v	Opening width (m)
H_v	Opening height (m)	z	Height from the opening lintel (m)
k	Extinction coefficient (m ⁻¹)	Z_n	Height of neutral plane (m)
l_x	Length along the EVF centerline, originating at the opening (m)	Greek	
m_f	Fuel mass (kg)	α_c	Convective heat transfer coefficient (W/m ² K)
\dot{Q}	Heat Release Rate (MW)	λ	Flame thickness (m)
q''	Heat flux to the façade (kW/m ²)	ε_z	Local emissivity of the flame (-)
Q_f	Fire load density (MJ/m ²)	ρ_{amb}	Air density at ambient conditions (kg/m ³)
RH	Relative humidity (%)	ρ_{500oC}	Air density at 500°C (kg/m ³)
T_{amb}	Ambient temperature (K)	σ	Stefan Boltzmann constant (5.67×10 ⁻⁸ W/m ² K ⁴)
T_f	Temperature of the flame (K)	φ_z	Configuration factor (radiation from EVF)
T_o	Temperature at the centre of the opening (K)	φ_f	Configuration factor (radiation from fire through windows)

1. INTRODUCTION

When a building fire is fully developed, flames may spill out of external openings, forming Externally Venting Flames (EVF), also known as façade fires. It is well established that EVF may significantly increase the risk of fire spreading to higher floors or adjacent buildings [1]. EVF may occur under both over-ventilated (OV) and under-ventilated (UV) fire conditions. During the initial stages of a compartment fire (pre-flashover stage), combustion is constrained at the interior of the compartment. When the fire is further evolved, flames in the ceiling jet may become long enough to eject from the compartment openings; in this case, EVF can be observed when the fire is still fuel-controlled. If the fire becomes ventilation-controlled (post-flashover stage), unburnt volatiles may eject from the opening; they are then mixed with ambient air and ignite, forming EVF.

In the case a fire erupts at the interior space of a building, it is possible for glass panes in windows to fail, thus forming compartment openings which increase the risk of EVF occurrence. Hazards associated with EVF are even greater in high-rise buildings. It is widely recognized that the fire behaviour of high-rise buildings is rather challenging in terms of fire safety as they involve some additional features compared to “conventional” low-rise building [1]. For example, combustible façade systems may pose an increased fire hazard during installation and construction prior to complete finishing and protection of such systems (e.g. Beijing Television Cultural Centre fire in 2009 and the Residential Building fire in 2010). Evacuation strategies in high-rise buildings are also a major safety issue. In addition, in high-rise buildings, as a part of energy efficient building techniques, there is an extensive use of external façade insulation wall systems such as Structural Insulation Panel Systems (SIPS), External Thermal Insulating Composite Systems (ETICS), Aluminium Composite Panels (ACP) and Metal Composite Material (MCM) claddings. Even though these systems may show superior energy-saving performance, in case they ignite it is possible to promote flame spread very fast and produce large amounts of gaseous toxic products. Table 1 reports a number of indicative recent high-rise building fires around the world, involving external fire spread via the building façade; this has also been the case in numerous other high rise building fires, highlighting the importance of understanding the mechanisms of fire spread due to EVF. In fire events where EVF are observed, external wall claddings are usually ignited, thus increasing the complexity of the observed fire spread mechanisms. For instance, in the CCTV building fire in 2009 [2] and the Marina Torch Tower fire in 2015 [3], the fire, counter-intuitively, was observed to spread downwards along the façade.

Façade fires represent 1.3 - 3.0% of the total number of building fires [4]. Building fires involving EVF appear to predominately occur in countries with poor regulatory controls concerning facades. The main fire safety aspects of such fire events concern façade heat flux and EVF plume characteristics [5], fire resistance of the façade assembly for load and non-load bearing structures and fire spread on the external surface or at the interior of a façade assembly [4]. Under-ventilated fire events generate the larger hazard in a structure regarding the heat flux impact and EVF development. The structural fire resistance of a façade assembly or a façade-floor junction can be assessed by performing standard fire resistance tests in conjunction with structural analysis depending on the properties of each façade component. In cases that the façade assembly contains combustible components, the additional heat due to fire spreading at the exterior surface of the façade should also be taken into account. However, the majority of current fire safety protection codes worldwide are lacking specific methodologies to evaluate the risks associated with EVF. For instance, the Eurocode design guidelines used across the European Union region, do not specifically address EVF-related risks. In order to implement specific actions towards EVF prevention, the following objectives should be met: protection against fire spread along the façade, maintaining the function of the fire compartmentation, protection against falling objects, reaction to fire requirements for components in the external wall and protection against fire spread between windows.

The key aspects of most fire safety regulations in relation to the fire performance of façades are mainly focused on the reaction to fire requirements for exterior wall assemblies and materials, fire stopping/barrier requirements for interior and exterior walls, separation distances of buildings and openings between stories, minimum separation distances of unprotected openings from a relevant boundary and requirements for sprinkler protection [4]. Nevertheless, aiming to develop a solid base for evaluation, testing and fire mitigation strategies for exterior façade systems potentially exposed to EVF, it is essential to fully comprehend the main phenomena characterising EVF.

In this context, the main aim of this work is to investigate the fundamental thermal phenomena governing EVF development and their impact on façade systems. Motivated by an increasing number of recent reports [4, 5, 6] suggesting that existing engineering design methodologies cannot describe with sufficient accuracy the thermal characteristics of EVF, this work emphasizes on the assessment of empirical correlations and design methodologies used to describe EVF and their impact on façade systems. Focusing on the thermal characteristics of EVF that affect the fire safety of a façade system, namely the EVF centreline temperature (T_z) and the EVF-induced heat flux to the façade (q''), the predictive accuracy of various design correlations is evaluated by means of comparison with medium- and large-scale fire tests conducted by the authors and found in literature.

Table 1. Indicative cases of recent high-rise façade fires.

Building	Location	Year	Type of façade system	Details
Ajman One residential cluster	Ajman, United Arab Emirates (UAE)	2016	Highly combustible plastic filled ACP	The fire erupted at a building in the Ajman One residential cluster of 12 towers and spread to at least one other tower, 1 injury, external fire spread [7]
Address Hotel	Dubai, UAE	2016	Highly combustible plastic filled ACP	Fire started on the 20 th floor of the building and only affected the exterior of the structure, 16 injuries, external fire spread [8]
Docklands Apartment Tower	Melbourne, Australia	2015	ACP	Fire started from an unextinguished cigarette on the sixth-floor balcony, no deaths or injuries, external fire spread [9]
Marina Torch Tower	Dubai, UAE	2015	Highly combustible plastic filled ACP	Fire started in the middle of the tower before spreading downwards, no deaths or injuries, external fire spread [3]
Residential Building	Grosny, Russia	2013	Ventilated façade	Fire started from a short circuit in an air-condition, no deaths or injuries, external fire spread [10]
Polat Tower	Istanbul, Turkey	2012	Ventilated façade	Fire burned through the building's external insulation, no deaths or injuries, external fire spread [10]
Al Baker Tower 4	Sharjah, UAE	2012	Highly combustible plastic filled ACP	Fire started at by a lit cigarette thrown on a balcony, no deaths or injuries, external fire spread [11]
Mermoz Tower	Roubaix, France	2012	Ventilated façade	Fire initiated at the second floor and spread rapidly upwards, 1 fatality, 10 injuries, external fire spread [10]
Wanxin Complex Fire	Shenyang, China	2011	ACP	Fire caused from explosive fireworks, external fire spread [12]
Residential Building	Dijon, France	2010	ETICS (EPS insulation, mineral wool fire barriers)	Arson fire started at the basis of the building from waste containers, 7 fatalities [10]
Residential Building	Shanghai, China	2010	ETICS (under construction)	Fire during renovation for installing exterior wall insulations, 58 fatalities, 71 injuries, external fire spread [12]
Television Cultural Centre (CCTV)	Beijing, China	2009	Ventilated façade (polystyrene insulation)	Fire caused from highly explosive fireworks at construction site on the roof - fire spread, 1 fatality, 7 injuries, external fire spread [2]

2. FIRE ENGINEERING DESIGN CORRELATIONS RELATED TO EVF

A detailed review and comparative assessment of the most widely used fire engineering design correlations, capable of describing the main characteristics of EVF that affect the fire safety of a building, has been performed previously by the authors [5]. These correlations can be used to estimate the geometrical and thermal characteristics of EVF, such as height, projection, width and centreline temperature, as well as EVF-induced heat flux on the façade. The semi-empirical correlations are commonly derived using simplified theoretical analyses in conjunction with experimental data [13, 14]. The correlations investigated in this work correspond to the state-of-the-art correlations currently available in the open literature. They are organised in two main categories, for the estimation of (a) EVF centreline temperature and (b) EVF-induced heat flux to the façade; their main characteristics, are briefly presented in the following sections.

2.1 EVF Centreline Temperature

A range of semi-empirical correlations to estimate the EVF centreline temperature rise above the ambient temperature as a function of height is shown in Table 2. In the majority of the investigated correlations, there is a strong dependence of the centreline temperature to the heat release rate. The height (z) used in the presented correlations corresponds either to the height above the opening spandrel (T1, T3 and T4) or the height above the virtual source (T2), estimated according to the methodology proposed by Yokoi [14]. In EN 1991-1-2 [15], different correlations are proposed when either No Forced Draught (NoFD) or Force Draught (FD) ventilation conditions are established; in the former case, openings are present only on one side of the fire compartment, whereas in the latter case, there are openings on opposite sides of the fire compartment or additional air is being fed to the fire from another source (e.g. mechanical ventilation). In Figure 1, the most important parameters used in the correlations, as well as the EVF shape assumed for NoFD and FD ventilation conditions, are depicted.

Table 2. Semi-empirical correlations for the estimation of the EVF centerline temperature.

Abbr.	Ref.	NoFD	FD
T1	[15]	$T_z = \left(1.0 - 0.4725 \frac{l_x W_v}{\dot{Q}}\right) (T_o - T_{amb})$	$T_z = \left(1.0 - 0.3325 \frac{l_x A_0^{1/2}}{\dot{Q}}\right) (T_o - T_{amb})$
T2	[16]	$(T_z + 273.15) = 24.6(\dot{Q}/1000)^{2/3} z^{-5/3}$	
T3	[14]	$T_z = \begin{cases} 2.0 \frac{\dot{Q}^{2/3}}{C_p^{2/3} \rho_{500^\circ C}^{2/3} g^{1/3} T_{amb}^{-1/3} (H_v - Z_n)}, & \text{when } \frac{z}{Z_n} \leq 0.3 \\ 2.0 \frac{\dot{Q}^{2/3}}{C_p^{2/3} \rho_{500^\circ C}^{2/3} g^{1/3} T_{amb}^{-1/3} (H_v - Z_n)}, & \text{when } \frac{z}{Z_n} > 0.3 \end{cases}$	
T4	[17]	$T_z = \begin{cases} 2.0 \frac{(\dot{Q}/1000)^{2/3}}{C_p^{2/3} \rho_{amb}^{2/3} g^{1/3} T_{amb}^{-1/3} (H_v - Z_n)}, & \text{when } \frac{z}{H_v - Z_n} < 0.64 \\ 1.6 \frac{(\dot{Q}/1000)^{2/3} z^{-1/2}}{C_p^{2/3} \rho_{amb}^{2/3} g^{1/3} T_{amb}^{-1/3} (H_v - Z_n)}, & \text{when } 0.64 \leq \frac{z}{H_v - Z_n} \leq 2.44 \\ 2.5 \frac{(\dot{Q}/1000)^{2/3} z^{-1}}{C_p^{2/3} \rho_{amb}^{2/3} g^{1/3} T_{amb}^{-1/3}}, & \text{when } \frac{z}{H_v - Z_n} > 2.44 \end{cases}$	

Fig. 1. Schematic of important parameters and assumed EVF shape, for NoFD (left) and FD (middle) conditions; front view of the compartment-façade configuration (right).

2.2 EVF-Induced Heat Flux Estimation

Aiming to estimate the EVF-induced heat flux on the façade, EVF are commonly modelled as a vertical plane adjacent to the façade; their heat flux levels are mainly influenced by the fire compartment geometry, heat release rate, ambient conditions (e.g. temperature, wind speed) and compartment temperature [18-21]. Generally, the heat balance for each point of a façade exposed to EVF can be expressed using Eq. (1).

$$\dot{q}'' = \dot{q}_{conv}'' + \dot{q}_{rad}'' = a_c(T_z - T_{wall}) + \varepsilon_z \varphi_z \sigma T_z^4 + \varphi_f \sigma T_f^4 - \sigma T_{wall}^4 \quad (1)$$

The local emissivity of the flame (ε_z) and the convective heat transfer coefficient (a_c) are commonly estimated using Eq. (2) and Eq. (3) or (4), respectively.

$$\varepsilon_z = 1 - \exp(-k\lambda) \quad (2)$$

$$a_c = c \left(\frac{\dot{Q}}{A_v} \right)^{0.6} \left(\frac{1}{d_{eq}} \right)^{0.4} \quad (3)$$

$$a_c = 9.8 \left(\frac{\dot{Q}}{17.85 A_v} + \frac{U}{1.6} \right)^{0.6} \left(\frac{1}{d_{eq}} \right)^{0.4} \quad (4)$$

In various heat flux calculation methodologies, including EN1991-1-2 [15], it is suggested to use predefined values for the extinction (k) and the convection heat transfer (a_c) coefficients for exposed and unexposed members, regardless of the type of the fuel, its burning rate and the EVF geometric characteristics. In the current work, the impact of various physical parameters that may actually influence the EVF's thermal impact to the façade, is investigated by means of comparison with available experimental data. Two main parameters are investigated, namely the effect of the fuel type on the extinction coefficient (k) and the the convective heat transfer coefficient (a_c). A key parameter analysis, presented in a previous work [5], revealed that the EVF shape does not significantly influence the estimated heat flux levels.

Four different models, namely HF1-HF4, presented in Table 3, are used for the estimation of the radiative and convective heat transfer components of the EVF-induced heat flux. In all the examined models, the total heat flux is estimated using Eq. (1), by assuming $\varphi_z = 1$ and neglecting the last two terms on the right hand side [5]; when relevant measurements were not available, the required EVF centreline temperature (T_z) was estimated using correlation T1.

Table 3. Investigated methodologies for the estimation of EVF-induced heat flux to the façade.

Abbr.	Radiative Heat Transfer	Convective Heat Transfer
	$k \text{ (m}^{-1}\text{)}$	$a_c \text{ (W/m}^2\text{K)}$
HF1	0.3	Eq. (3)
HF2	k_{fuel} (Table 4)	Eq. (3)
HF3	0.3	25 [15]
HF4	0.3	Eq. (4), $1/d_{eq} = 1.0$

Model HF1 serves as a base-case scenario, by assuming a constant k value (0.3) and using Eq. (3) to estimate a_c . Aiming to improve the accuracy of the EVF-induced heat flux estimations, the effect of using fuel-dependent extinction coefficient values is investigated in model HF2. The k_{fuel} values used for each test case, based on the actual fuel employed in the respective fire test, are given in Table 4. The presented bibliographic values are indicative and may provide practical engineering estimates. If more accurate estimations are required, a thorough numerical simulation analysis should be performed, by employing a more rigorous heat transfer methodology. Model HF3 investigates the 25 W/m²K value proposed in EN1991-1-2 for a_c , whereas in model HF4, a different correlation, Eq. (4), is used for the estimation of a_c .

Table 4. Extinction coefficient or monochromatic absorption coefficient for various fuels.

Fuel	$k_{fuel} \text{ (m}^{-1}\text{)}$	Ref.
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Propane	13.32	[22]
City gas/Methane	6.45	[22]
Wood cribs	0.80	[22, 23]
Assorted furniture	1.13	[22]

3. FUNDAMENTAL PHENOMENA AFFECTING EVF

It is well established that during a fire event, one of the weakest links in a building is its window glass. Due to the thermal stresses, normal glass may crack and fall out when exposed to relatively low temperatures and radiant heat fluxes [24]. In a fully developed fire, flames may spill out of external openings (e.g. windows) in case the glazing fails. Fire compartment and opening geometry and the prevailing ventilation conditions are the most significant parameters influencing the EVF geometric and thermal characteristics [5, 6, 13, 15, 18, 19, 25]. During the initial stages of the fire, compartment geometry is important since enclosure dimensions are decisive on how close the initial fire is located in relation to other combustible materials, ventilation openings and compartment boundaries. Openings severely impact the fire behaviour because as soon as flaming combustion occurs the fire becomes dependent on oxygen availability in order to maintain itself; they also control fire growth rate and compartment temperature. If a wall exists above the opening through which the EVF emerges, the temperature difference between the fire plume and ambient air creates a strong buoyant current that causes EVF to move upwards. Horizontal projections and spandrels have also been found to greatly influence the EVF characteristics [20]. Recently, the effect of balconies on the fire spread via external windows into upper floors was experimentally and numerically investigated [26]. The absence of a balcony between windows of successive floors allows EVF to move along the façade; in case there is a balcony, the risk of fire spread into the upper floors is reduced. However if the balcony is of the same width as the opening, there is a high lateral EVF spread and as a result the limiting of the EVF spread is not as effective. Additionally, ventilation conditions, such as Forced Draught (FD) and No Forced Draught (NoFD) modes play an important role in EVF development [5, 18, 19]. When external wind is blowing parallel to the building façade, EVF will eventually deflect horizontally to the side of the opening. This may affect openings on the adjacent compartments or even compartments located in higher levels.

Table 5. Review on fundamental experimental (Exp.), numerical (Num.) and theoretical (Th.) studies on EVF and their effects on the façade.

No.	Ref.	Exp.	Num.	Th.	Scale		Short description
					Medium	Large	
1	[21]	×	×	✓	✓	×	Revisiting EVF physics and suggesting two new length scales for over- and under-ventilated fires
2	[27]	✓	✓	×	×	✓	Formation and validation of a simplified model for heat flux estimation due to EVF
3	[17]	✓	×	×	✓	×	Formation and validation of a model for EVF prediction with emphasis on wall attachment
4	[29]	✓	×	×	✓	×	Investigation of ventilation conditions effect on EVF and heat fluxes at the facade
5	[18]	✓	✓	×	✓	×	Full scale study of the EVF associated phenomena under realistic fuel loads
6	[30]	✓	×	×	×	✓	Formation of a model for estimation of heat release due to combustion of excess fuel in EVF
7	[20]	✓	×	✓	×	✓	Investigation of façade exposure due to EVF. Identification of parameters influencing thermal exposure
8	[13]	✓	×	✓	×	✓	Formation and validation of a set of

							correlations to predict the effects of EVF on the heating of external structural elements
9	[30]	✓	×	×	×	✓	Investigation of EVF projection and correlation with fuel load
10	[31]	✓	×	×	×	✓	Investigation of EVF physical characteristics
11	[14]	✓	×	✓	✓	✓	Investigation of EVF physical characteristics

Research on EVF has been carried out since the 1960s as illustrated in Table 5. A number of medium- and large-scale fire tests combined with numerical simulations have proven useful in the identification of the physical aspects of EVF and of the parameters affecting their development. Initial research efforts focused on the identification and characterization of the main EVF physical characteristics and their dependency on fuel load and the geometrical characteristics of the compartment [14, 30, 31]. The first correlations used to describe the EVF physical characteristics were developed in the 1960s by Yokoi [14] and further evaluated and improved in the 1980s by Law [13]. At a later stage, Oleszkiewicz [26] pointed his research towards EVF exposure of facades, by identifying the main parameters influencing the induced thermal exposure. The first thorough study on the effects of ventilation conditions, except of the initial work of Law [13], was conducted by Klopovic [18, 19] and Huang [28]. The Law model was revisited by Empis in her well-grounded work on the parameters influencing the heat flux incident on facades [27]. Later on, a number of models for predicting EVF characteristics and their impact on facades have been developed by other researchers [17, 29]. More recently, Lee and co-workers [21] revisited the EVF physics by introducing two new length scales to describe EVF evolution in over-ventilated (OV) and under-ventilated (UV) conditions.

4. EXPERIMENTAL INVESTIGATION OF EVF

4.1 Medium-scale façade fire test

In the frame of the current study, a series of fire tests was conducted in a medium-scale compartment-façade fire facility. The compartment was a ¼ scale model of an ISO 9705 room [32]. The internal compartment dimensions were 0.60 m × 0.90 m × 0.60 m; the external facade wall measured 0.658 m × 1.8 m. A double layer of 0.0125 m thick fire-resistant gypsum plasterboards was used as the internal (compartment) and external (façade) lining material. The fire compartment opening, corresponding to an open door, located at the middle of the northern wall, measured 0.20 m × 0.50 m. A range of realistic fire scenarios, relevant to building fires, was developed for the EVF measurement campaign. The majority of the actual furniture found in contemporary residential environments consists of hydrocarbon-based thermoplastic materials, including plastics and foams that, when ignited, melt and burn similar to liquid-fuel pool fires. In this context, aiming to simulate realistic building fire conditions, an “expendable” liquid fuel (n-hexane) pool fire source was employed. Recording of the dynamic behaviour of the EVF was carried out using a selectively distributed network of sensors that allowed monitoring of important physical parameters, such as flame envelope geometry, gas and wall surface temperatures, façade heat flux, fuel mass loss and gas species concentrations. A total of 102 thermocouples (1.5 mm K-type) has been employed to record the temporal evolution of gas and solid surface temperatures; the EVF shape was estimated using an in-house developed image processing tool [6]. A schematic of the experimental facility, illustrating the locations of the employed measuring devices, is given in Fig. 2 [33].

A thorough repeatability analysis has been presented in a previous work [33], by performing three identical tests in the same medium-scale façade fire test facility; it has been demonstrated that good levels of experimental repeatability are achieved. A parametric study was performed, by varying the total fuel load (test cases 1, 2 and 3) and the opening dimensions (test case 4). The fire load used in test cases 2 and 4 was identical; the former case corresponds to a “door” opening, whereas the latter case refers to a “window” opening. A summary of the main operational parameters, i.e. initial fuel mass (m_f), fire load density (Q_f), opening height (H_v) and width (W_v) and average total heat release rate (\dot{Q}) for all the examined test cases is given in Table 6.

Fig. 2. General layout (left) and characteristic image (right) of a medium-scale façade fire test.

The average heat release rate (\dot{Q}) has been calculated by means of the experimental fuel consumption rate and the lower heating value of n-hexane (43521 kJ/kg), estimated using an isoperibolic bomb calorimeter. Test cases 2, 3 and 4 corresponded to under-ventilated fire conditions [6]. Experimental results suggest the existence of three characteristic EVF phases, namely “Internal Flaming” (IF) corresponding to the initial period when combustion is limited at the interior of the fire compartment, “Intermittent Flame Ejection” (IFE), when flame jets appear intermittently outside the compartment and the “Continuous External Flame” (CEF) period that essentially spans the time period when EVF are consistently ejecting through the opening [33].

Table 6. Summary of main operational parameters for the examined test cases.

Test Case		1	2	3	4
m_f	(kg)	0.655	1.539	3.078	1.539
\dot{Q}_f	(MJ/m ²)	53.18	125.0	250.0	125.0
H_v	(m)	0.5	0.5	0.5	0.3
W_v	(m)	0.2	0.2	0.2	0.2
\dot{Q}	(kW)	79.0	207.0	233.0	105.0

4.2 Large-scale façade fire test

Aiming to further investigate the fundamental characteristics of EVF, a large-scale fire test was performed at the premises of the Greek Fire Academy [34]. A timber frame compartment was lined with two layers of 0.0125 m fire resistant gypsum plasterboards. The internal dimensions of the test compartment measured 1.760 m × 0.800 m × 2.100 m (Fig. 3). The compartment exhibits a single opening (window), measuring 0.765 m × 1.100 m. The external façade wall measured 2.614 m × 5.230 m. The window is located on the south side; the distance of the window sill from the compartment’s floor is 0.940 m. A ventilated façade system was installed on the south side of the compartment. The internal façade surface was formed using commercial 0.015 m thick gypsum plasterboard; timber studs and battens were used to support the façade on top of the compartment. The external cladding panels, comprising 0.0125 m thick cement boards covered by a 0.005 m thick layer of plaster coating, were supported using perforated steel studs the width of the air cavity formed between the two layers is 0.025 m. An extensive set of sensors was installed both inside and outside the test compartment, aiming to record the temporal variation of several important physical parameters, such as gas and wall surface temperatures, gas velocities and mass loss rate. Emphasis was given to the characterization of the temperature environment adjacent to the façade wall along the height of the EVF plume.

A stainless steel rectangular pan, measuring 0.700 m × 0.700 m × 0.250 m, was installed at the geometrical centre of the room, 0.1 m above the compartment floor, holding the 56.7 kg of liquid n-hexane used as fire load. The lower heating value of the n-hexane used in the tests was estimated, using an isoperibolic oxygen bomb calorimeter, to be 43521 kJ/kg. The fuel mass was continuously monitored using a load cell, installed under the pan. This "expendable" fuel source was employed to better simulate realistic building fire conditions. The fire load and opening dimensions were selected in order to establish strongly under-ventilated fire conditions, thus ensuring the development of an EVF. The peak fire power achieved, estimated using the instantaneous fuel mass loss rate, was 2.76 MW.

Fig. 3. General layout (left) and characteristic image (right) of the large-scale compartment-façade fire test.

4.3 Additional medium- and large-scale façade fire tests

Aiming to validate and comparatively assess the fire engineering design correlations and methodologies presented in Section 3, predictions are compared to EVF centreline temperatures measurements obtained in the aforementioned medium-scale (Section 4.1) and large-scale (Section 4.2) façade fire tests. In order to

further broaden the scope of the validation study for the investigated empirical correlations and models, an additional large set of measurements, obtained in various large-scale fire tests found in the literature, was also used. The main characteristics of each fire test, such as compartment geometry, ventilation characteristics and fire power, are presented in detail in Table 7. The ventilation regimes for each test case are also tabulated; the majority of the cases correspond to ventilation-controlled fires (UV).

Table 7. Main characteristics of the medium- and large-scale fire tests used in the validation study.

Test Case	Ref.	Scale	$W \times D \times H$ (m ³)	A_o (m ²)	$A_{o,FD}$ (m ²)	Vent. Cond.	\dot{Q} (MW)	t_{dur} (min)	Vent. Regime
1	[33]	Medium	$0.6 \times 0.9 \times 0.6$	0.2×0.5	-	NoFD	0.079	6.2	OV
2	[33]	Medium	$0.6 \times 0.9 \times 0.6$	0.2×0.5	-	NoFD	0.207	3.9	UV
3	[33]	Medium	$0.6 \times 0.9 \times 0.6$	0.2×0.5	-	NoFD	0.233	9.9	UV
4	[33]	Medium	$0.6 \times 0.9 \times 0.6$	0.2×0.3	-	NoFD	0.105	10.9	UV
5	[34]	Large	$1.7 \times 0.8 \times 2.1$	0.8×1.1	-	NoFD	2.76	16.0	UV
6	[18,19]	Large	$5.3 \times 3.6 \times 2.4$	2.4×1.5	-	NoFD	6.34	32.0	UV
7	[18,19]	Large	$5.3 \times 3.6 \times 2.4$	2.4×1.5	0.8×2.0	FD	5.03	32.0	UV
8	[20]	Large	$5.9 \times 4.4 \times 2.8$	2.6×2.7	-	NoFD	10.3	30.0	OV
9	[27]	Large	$3.6 \times 4.8 \times 2.5$	2.4×1.2	0.9×1.9 0.9×2.0	FD	8.8	19.0	UV
10	[35]	Large	$3.0 \times 4.3 \times 1.7$	2.0×1.00	-	NoFD	2.8	5.0	UV
11	[35]	Large	$3.0 \times 4.3 \times 1.7$	2.0×1.2	0.5×0.6 ($\times 4$)	FD	4.2	5.0	OV

5. RESULTS AND DISCUSSION

5.1 EVF Centreline Temperature

A comparison of EVF centreline temperature measurements to predictions obtained by using correlations T1-T4 are shown in Fig. 4, where the vertical distribution of measured and predicted time-averaged centreline EVF temperatures are depicted. Correlation T1 is applied only in test case 3, since in the rest of the cases the total fire power was lower than the correlation's range of applicability. Under low fire load conditions (e.g. test case 1) correlations T2 and T3 are found to over-predict experimental data, whereas correlation T4 shows a remarkable agreement with measured values. In cases of increased fire load (e.g. test case 3), correlations T1 and T4 considerably under-predict the measured values, whereas correlations T2 and T3 over-predict the experimental data. The point heat source assumption employed in correlation T2 results in a good qualitative agreement with the measured centreline temperatures in test cases 1, 2 and 4. Nevertheless, predictions of correlation T2, which is based on the experimental investigation of fire plumes (considered as upward hot currents), do not agree quantitatively with the actual EVF centreline temperature profile, especially near the opening. Yokoi's methodology, correlation T3, over-predicts the experimental data in test cases 1, 2 and 3, but appears to accurately estimate temperatures near the opening for all test cases. Correlation T4 shows good qualitative and quantitative agreement in test cases 1, 2 and 4 but slightly under-predicts the measured values in test case 3. Generally, the observed discrepancies between experimental data and correlations may be attributed to the fact that the performed experiments resulted in continuous and consistent EVF, whereas literature reports suggest that the majority of correlations originate from temperature measurements in the fire plume region. Overall, the use of T1 correlation is found to under-estimate the experimental values, but no safe conclusion can be derived since it was only applied in one medium-scale test case. The rest of the correlations, namely T2, T3 and T4, may be safely used (conservative predictions), although they exhibit reduced accuracy in positions near the top of the opening.

Fig. 4. Vertical distribution of measured [38] and predicted time-averaged centreline EVF temperatures; effect of fire load (left) and opening factor (right).

In Fig. 5, predictions of the vertical distribution of EVF centreline temperatures using correlations T1-T4 (Table 3) are compared to experimental data obtained in the large-scale fire test of Klopovic and Turan [18, 19], and a ventilated façade large-scale fire test [34] performed by the authors. Under No Forced Draught (NoFD) conditions (test case 5), correlation T1 is consistently under-predicting the actual centreline temperature, whereas correlations T3 and T4 exhibit a more conservative behaviour (over-prediction); correlation T2 demonstrates a rather inconsistent behaviour. On the other hand, when Forced Draught (FD) conditions are established (test case 7), only correlation T1 is capable of consistently over-estimating the centreline temperature. Correlations originating from the experimental investigation of fire plumes or upward moving hot jets, such as T2 and T4, significantly under-predict the EVF centreline temperature near the opening.

Fig. 5. Vertical distribution of measured [19, 20, 39] and predicted centreline EVF temperatures for test cases 5 (left) and 7 (right).

5.2 Heat Flux on the Exposed Façade Surface

The temporal evolution of the measured and estimated heat flux at the exposed façade surface in the medium-scale façade fire tests are illustrated in Fig. 6; the latter values are estimated using the models presented in Table 3. A typical behaviour of an under-ventilated compartment fire can be observed which is characterized by 3 distinct phases that appear in succession [35].

Fig. 6. Vertical distribution of measured and predicted EVF-induced heat flux on the exposed façade surface; effect of fire load (left) and opening factor (right).

Initially, combustion is constrained in the interior of the fire compartment (IF period) and in the vicinity of the fuel pan an advection stream is created. Gradually, the flame front moves away from the fuel pan, expanding radially and horizontally towards the opening. In that phase, external flame jets and quick flashes appear at the exterior of the fire compartment, signifying the beginning of the IFE stage. As time passes, CEF is observed due to the sustained external combustion of unburnt volatiles, during the quasi-steady phase of fully developed fire. Throughout the latter phase, EVF consistently covers the region above the opening resulting in higher values of heat flux in the façade surface.

Model HF1 is used as a base case; it is found to under-predict the experimental data. Model HF2, which takes into account the specific fuel properties for the estimation of the extinction coefficient, results in predictions that err on the safe side, with the exception of test case 4 where a slight under-prediction is observed (approximately 10%). Less conservative estimations are derived when model HF3 is employed, where a constant value of convective coefficient is used, as proposed in EN 1991-1-2 [15]. When a more rigorous methodology is used to estimate the convective coefficient (model HF4), results are not significantly improved. Based on the aforementioned observations, it appears that the most important influencing parameter is the effect of fuel on the extinction coefficient value (model HF2).

Aiming to further investigate the applicability of the examined fire engineering design correlations, predictions are also compared to the EVF-induced heat flux measurements obtained in 6 large-scale compartment-façade fire tests found in the literature [5]. Table 8 presents a summary of the estimated relative errors for all the examined test cases; positive values indicate “over-prediction” (conservative design values), whereas negative values suggest “under-prediction” (non-conservative design values). Predictions of the heat flux to the façade, in both medium- and large-scale configurations, using various methodologies highlighted the importance of the extinction coefficient (k_{fuel}). When model HF2 is used, predicted values generally err on the safe side, under both NoFD and FD conditions. In the case of FD conditions, more conservative predictions are obtained using model HF3, where a constant value for the convective heat transfer coefficient is employed. An attempt to use a more rigorous methodology for the calculation of the convective heat transfer coefficient has not been successful, as demonstrated by the large errors obtained when method HF4 is used. Overall, model HF2, where the effect of the fuel type used in

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Figure captions:

Fig. 1. Schematic of important parameters and assumed EVF shape, for NoFD (left) and FD (middle) conditions; front view of the compartment-façade configuration (right).

Fig. 2. General layout (left) and characteristic image (right) of a medium-scale façade fire test.

Fig. 3. General layout (left) and characteristic image (right) of the large-scale compartment-façade fire test.

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