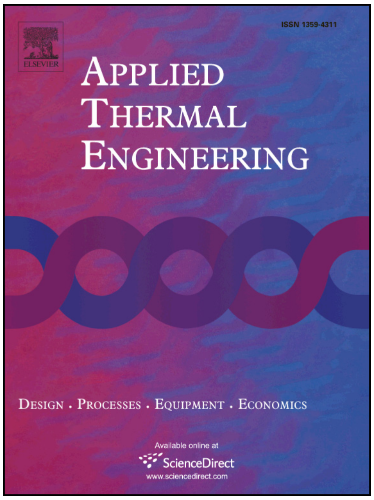
Title	A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes
Type	Article
URL	https://clok.uclan.ac.uk/19602/
DOI	https://doi.org/10.1016/j.applthermaleng.2016.08.068
Date	2016
Citation	Fsadni, Andrew, Whitty, Justin and Stables, Matthew (2016) A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes. Applied Thermal Engineering, 109 (Part A). pp. 334-343. ISSN 1359-4311
Creators	Fsadni, Andrew, Whitty, Justin and Stables, Matthew



A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes

Andrew Michael Fsadni, Justin P.M. Whitty, Matthew A. Stables

PII: S1359-4311(16)31422-3
DOI: http://dx.doi.org/10.1016/j.applthermaleng.2016.08.068
Reference: ATE 8867

To appear in: *Applied Thermal Engineering*

Received Date: 10 June 2016
Revised Date: 10 August 2016
Accepted Date: 11 August 2016

Please cite this article as: A.M. Fsadni, J.P.M. Whitty, M.A. Stables, A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes, *Applied Thermal Engineering* (2016), doi: http://dx.doi.org/10.1016/j.applthermaleng.2016.08.068

Title: A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes

Authors: Andrew Michael Fsadni*, Justin P.M. Whitty, Matthew A. Stables

*Corresponding author

Contact details:

Address: University of Central Lancashire, School of Engineering, Rm. KM124, Preston, UK, PR1 2HE

Email: afsadni@uclan.ac.uk

Tel: +44 1772893812

A brief review on frictional pressure drop reduction studies for laminar and turbulent flow in helically coiled tubes

Abstract

This review, summarises the pertinent literature on drag reduction (DR) in laminar and turbulent flow in coiled tubes. Due to their compact design, ease of manufacture and superior fluid mixing properties, helically coiled tubes are widely used in numerous industries. However, flow through coiled tubes yields enhanced frictional pressure drops and thus, drag reduction is desirable as it can: decrease the system energy consumption, increase the flow rate and reduce the pipe and pump size. The main findings and correlations for the friction factor are summarised for drag reduction with the: injection of air bubbles and addition of surfactant and polymer additives. The purpose of this study is to provide researchers in academia and industry with a concise and practical summary of the relevant correlations and supporting theory for the calculation of the frictional pressure drop with drag reducing additives in coiled tubes. A significant scope for future research has also been identified in the fields of: air bubble and polymer drag reduction techniques.

Keywords: Helically coiled tube, drag reduction, frictional pressure drop, surfactants, polymer solutions.

1. Introduction

Due to their compact design, ease of manufacture and high efficiency in heat and mass transfer, helically coiled tubes are widely used in a number of industries and processes such as in the food, nuclear, aerospace and power generation industries and in heat recovery, refrigeration, space heating and air-conditioning processes. Due to the formation of a secondary flow, which inherently enhances the mixing of the fluid, helically coiled tube heat exchangers are known to yield improved heat transfer characteristics when compared to straight tube heat exchangers. The secondary flow, which finds its origins in the centrifugal force, is perpendicular to the axial fluid direction and reduces the thickness of the thermal boundary layer. However, for single and multiphase flows, the secondary flow yields a substantial increase in the frictional pressure drop, which often results in diminished system efficiencies (due to enhanced pumping power requirements). For air-water two-phase bubbly flow in helically coiled tubes, Akagawa *et al.* (1971) reported frictional pressure drops in the range of

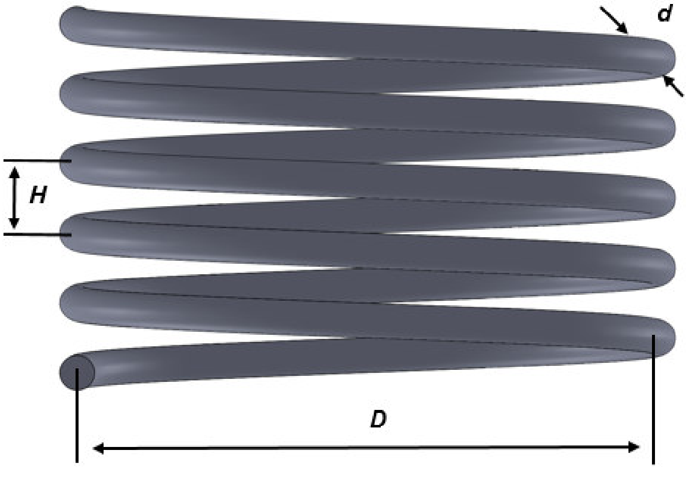


Figure 1: Schematic representation of helical pipe characteristics.

1.1 to 1.5 times greater than those in straight tubes, ceteris paribus, whilst, with the use of nanofluids, such a penalty could nullify the enhanced efficiencies gained with the dispersion of nanoparticles in the base fluid (Aly, 2014). Moreover, due to the secondary flow, the flow characteristics are significantly different to those in straight tubes. Whereas in straight tubes the transition from laminar to turbulent flow occurs at Reynolds numbers in the region of 2500, the transition in curved tubes takes place at higher Reynolds numbers. The critical Reynolds number (Eq. (1)) is used to determine the transition of the flow from laminar to turbulent flow (Ito, 1959).

$$Re_{crit} = 2E4\delta^{0.32} \quad \textbf{(1)}$$

where δ is the curvature ratio defined through Eq. (2).

$$\delta = \frac{d_t}{D_c} \quad \textbf{(2)}$$

For δ^{-1} <8.6E2 whilst for δ^{-1} >8.6E2, Re_{crit} for a curved tube is equal to that for a straight pipe.

Another dimensionless number, unique to coiled tubes, is the Dean number, given in Eq. (3). It is used to characterise the flow in curved tubes and quantifies the magnitude of the secondary flow due to the centrifugal force (Mohammed and Narrein, 2012).

$$De = Re\sqrt{\delta} \quad \textbf{(3)}$$

The performance of coiled tubes is a complex function of the coil design parameters (Fig. 1) as well as the resultant pressure drop. Therefore, drag reduction (DR) techniques could be particularly beneficial for systems with curved tubes. Intriguingly, whilst numerous investigations have been reported on DR in straight channels and pipelines with the: injection of air bubbles (Nouri *et al.*, 2013; Fujiwara *et al.*, 2004), dispersion of surfactants (Gasljevic and Matthys, 1997) and polymers (Wei and Willmarth, 1992; Al-Sarkhi and Hanratty, 2001), there is a paucity of research in the field of curved tubes. Moreover, researchers have reviewed the frictional DR techniques in straight channels and pipes (Merkle and Deutsch, 1992; Al-Sarkhi, 2010; Murai, 2014) whilst the sole study that reviewed DR in curved tubes was presented by Broniarz-Press *et al.* (2007). However, the latter focussed on the application of DR surfactant and polymer additives and hence, did not provide a holistic review of the relevant studies. The aim of the current study is to critically review the experimental and numerical studies done on DR in single-phase (water) laminar and turbulent flow through coiled tubes. Such studies are categorized in three sections, representing the pertinent techniques reported. Moreover, this paper complements the earlier review undertaken by the authors of the present study (Fsadni and Whitty, 2016), as it further elucidates the underpinning physics of air-water bubbly flow through curved tubes. It is the authors' hope that this review will be useful to both academics and industry based engineers through the provision of a concise report on the relevant current knowledge.

2. Injection of air bubbles

Over the past 40 years, the injection of microbubbles in the turbulent boundary layer has been investigated by numerous investigators, with the first study reported by McCormick and Bhattacharyya (1973) who investigated the DR to a submersible hull. As summarised in Table 1, Shatat *et al.* (2009a&b) were the first to investigate DR with the injection of air bubbles in laminar and turbulent low through helically coiled tubes. They reported a diminished DR efficiency (Eq. (4)) over that of straight tubes. Such results were more significant with higher curvature ratios whilst, the DR increased with higher air volumetric void fractions (*VF*) and decreased with higher *Re* numbers (Fig. 2). Moreover, DR was limited to turbulent flow. Similar results were reported by Saffari *et al.* (2013) who measured a 25% DR at a *VF* of 0.09 in turbulent flow bubbly flow. The latter study did not investigate the DR with straight tubes. However, their experimental parameters are comparable to those used by Nouri *et al.* (2013) who reported a DR of 35% for a *VF* of 0.09 in straight tubes.

$$DR = 100\left(\frac{f_l - f_{tp}}{f_l}\right) \quad \textbf{(4)}$$

where f_l is the Fanning friction factor for single-phase flow and f_{tp} is the friction factor for two-phase flow.

For a straight vertical pipe, Fujiwara *et al.* (2004) reported that, with a high *VF* in the near-wall region, the turbulence intensity and Reynolds stress are reduced in a wide region of the pipe. The turbulence energy dissipation occurs around the bubbles due to bubble-induced eddies, whilst the diminished fluid density in the near-wall region reduces the shear stress, thus resulting in a lower system frictional pressure drop. Saffari *et al.* (2013) reported that in curved tubes, higher *Re* numbers and curvature ratios, result in larger centrifugal forces which force the lower density phase (air bubbles) to migrate towards the inner tube wall region. Resultantly, the shear stress at the inner tube wall region is lower than that at the outer wall region. Hence, the uneven distribution of the air bubbles at higher *Re* numbers and curvature ratios results in a diminished DR efficiency.

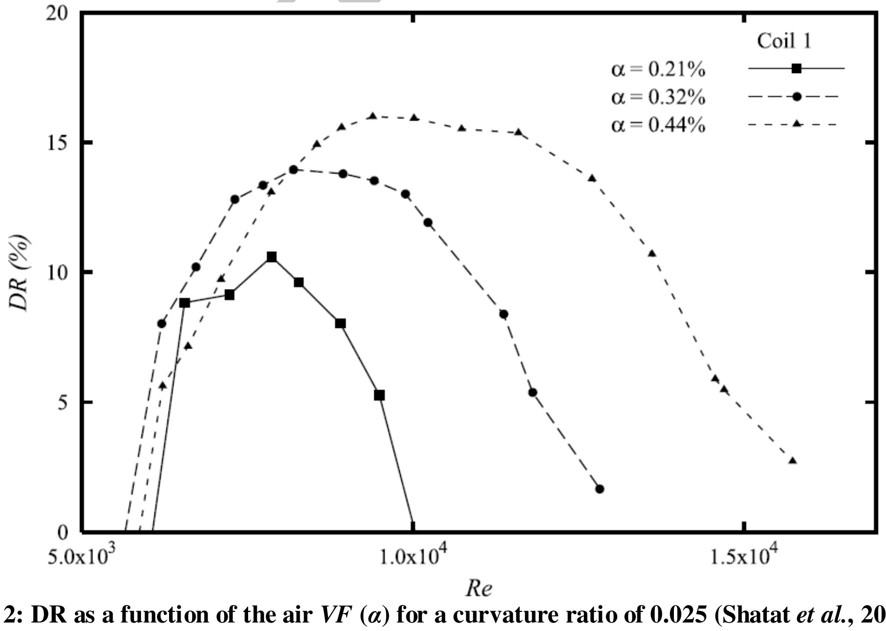


Figure 2: DR as a function of the air *VF* (α) for a curvature ratio of 0.025 (Shatat *et al.*, 2009a. Fig. 11).

These studies are in a general agreement with relevant theory and numerous DR studies reported for channel and straight tube flow. Moreover, there is significant scope for further research in DR (in coiled tubes) as a function of the bubble diameter. In fact, for straight tubes and channels, some controversy surrounds the impact of bubble size on the DR, where some investigators reported the DR to be a strong function of the bubble diameter (Liu 1993; Murai *et al.*, 2007) while other investigators reported the DR to be independent of the bubble diameter (Moriguchi and Kato, 2002; Shen *et al.*, 2006). The relation of the bubble induced DR studies with those reviewed for two-phase gas-liquid frictional pressure drop characteristics in coiled tubes (Fsadni and Whitty, 2016) remains indeterminate. In fact, the latter investigations reported a general agreement with the Lockhart and Martinelli correlation for straight tubes, with the two-phase frictional pressure drop multiplier in excess of unity.

3. Surfactant additives

Surface-active agents (surfactants) are low molecular weight, viscous, non-polymer, water-based chemicals that tend to accumulate at a surface and diminish interactive forces between the molecules of the base fluid, thus reducing the surface tension. Inaba *et al.* (2005) reported that surfactant additives form a network structure of rod-like micelles which absorbs the turbulent energy with its flexibility and deformation, thus leading to a flow laminarisation effect. Hence, surfactants enhance the elastic properties of the fluid with the resultant increase in DR. Unlike polymer based fluids, the mechanical degradation of the micelle network at high shear stresses is completely reversed at a low flow rate. All the studies reviewed reported a DR limited to the transition and turbulent flows, with a reduced DR in curved tubes when compared to straight tubes, ceteris paribus. Such findings were attributed to the formation of the secondary flow which is largely unaffected by the surfactant additive. Gasljevic and Matthys (1999) reported that for a velocity range of 2-5m/s, the secondary flow effects were separated from the turbulence effects through the use of the turbulence reduction – drag (TRD) method given in Eq. (5). This yielded a TDR of 70% (turbulence suppression) for both coiled and straight tubes (Fig. 3). In contrast, Broniarz-Press et al. (2003) reported that the tube curvature effect on the friction factor was diminished due to the damping of the secondary flows streams. A broad analogy can be made with nanofluid flow in coiled tubes where, nanoparticles were also attributed to the mitigation of the secondary flow (Fsadni and Whitty, 2016).

$$TRD = \frac{FC_{bf,tb} - FC_{DRF}}{FC_{bf,tb} - FC_{bf,lm}} \qquad \textbf{(5)}$$

where *lm* refers to the laminar flow of the base fluid (without the DR additives) at the same *Re* number and *tb* refers to the turbulent flow of the base fluid.

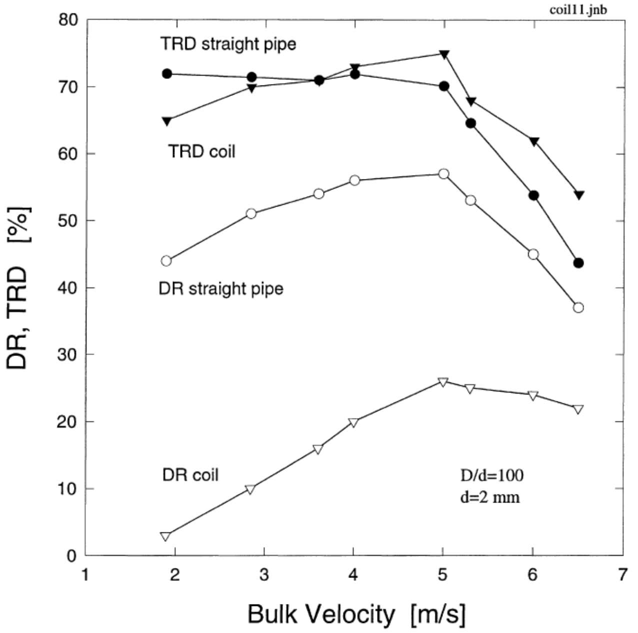


Figure 3: Friction reduction in terms of *DR* and *TRD* for a coiled and straight pipe (Gasljevic and Matthys, 1999 Fig. 4).

At laminar flow conditions, Weber *et al.* (1991) and Gasljevic and Matthys (2009) reported an increase in the frictional pressure drop (compared to water). This was attributed to the enhanced solution viscosity. There is a general agreement amongst the studies reviewed that lower coil curvatures and higher surfactant concentrations yielded higher DR efficiencies. Moreover, Kamel and Shah (2013) reported that at higher concentrations, surfactant solutions are more resistant to mechanical degradation and hence, yield higher DR efficiencies at increased *Re* numbers. Therefore, Broniarz-Press *et al.* (2002) reported that DR is a strong function of the surfactant concentration, with DR evident above a critical concentration. Inaba *et al.* (2005) reported that the dynamic nature of surfactant DR additives render them particularly relevant for heating systems. However, such comments should be considered in light of the fact that these additives are known to yield reduced heat transfer coefficients. Kostic (1994) attributed this phenomenon to the non-homogenous turbulence resulting from the flow-induced anisotropicity of the highly structured micelle network. Weber *et al*. (1991), Inaba *et al*. (2000&2005), Aly *et al*. (2006) and Kamel and Shah (2013) presented correlations for the calculation of the friction factor in surfactant solutions. Due to the Non-Newtonian properties of these solutions (C>3,000ppm), correlations were developed as a function of the modified or generalised *Re* and *De* numbers.

4. Polymers additives

Toms (1948) reported that the addition of minute concentrations of high-molecular weight, long chain and flexible polymers to a Newtonian solvent can yield significant DR properties. Whilst it is widely accepted that the DR efficiency is a strong function molecular weight and distribution, molecular structure and solubility, the underpinning physics are known to be complex and not well-understood (Gallego and Shah, 2009). Factors such as shear thinning, viscoelasticity and molecular stretching have been suggested to diminish the turbulence in the fluid (Bird *et al.*, 1987), thus resulting in DR.

Shah and Zhou (2001) stated that the DR mechanism of polymers occurs at the boundary layer and therefore is typically more effective in smaller tube diameters. Moreover, in agreement with the findings reported for air-bubble injection, DR efficiency decreases with higher coil curvatures. This is inherent to the effects of the centrifugal force on the fluid flow. DR is also a function of the ability of the polymer to resist thermal and mechanical degradation. Shah *et al.* (2006) reported that at a volume concentration of 0.07%, the widely used partially hydrolysed polyacrylamide (PHPA) copolymer (Nalco ASP-820) yielded the highest DR (65%). At this concentration, it was assumed that the fluid behaviour is quasi-Newtonian. This concentration was subsequently used by Gallego and Shah (2009) and Ahmed Kamel (2011). Gallego and Shah presented a unique generalised friction pressure correlation for DR polymer solutions in coiled tubes. Their correlation assumed that the appropriate characteristic polymer solution viscosity is relative to the zero shear rate viscosity, that is, the shear stress required to deform the polymer molecule from its equilibrium state.

The effect of the polymer concentration is also function of the specific physical conditions of the flow. Resultantly, Shah and Zhou (2001) reported that for large tubes and low flow rates, high concentrations of polymer additives increased the fluid drag and delayed the onset of DR (Fig. 4). For small diameter tubes, the opposite effect was reported and thus, a higher polymer concentration increased the DR.

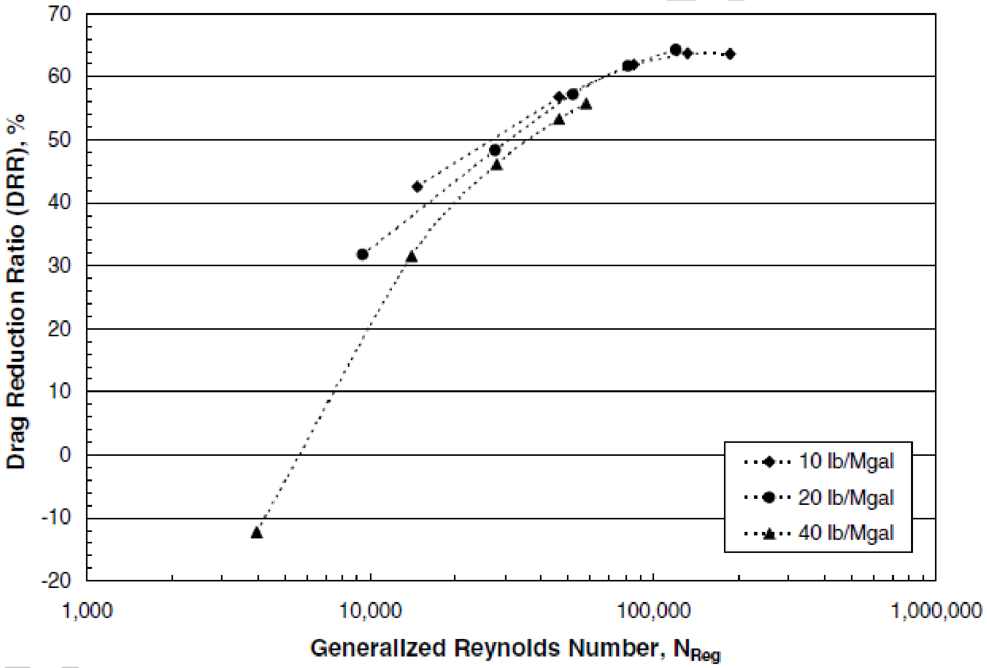


Figure 4: Effect of polymer concentration (Xanthan) on DR ratio (Shah and Zhou, 2001 Fig. 5).

The effect of elevated temperatures on the DR of polymers in coiled tubes was investigated by Gallego and Shah (2009) and Ahmed Kamel (2011) who reported that, in contrast to the findings for straight tubes, DR remained quasi-constant (Ahmed Kamel) or increased (Gallego and Shah) with temperature. It is widely accepted that with polymer solutions in straight tubes, elevated temperatures yield a drop in the DR. This is due to a combination of factors, such as the deterioration of the solvent-polymer interaction and the diminishing of the macromolecule size (Clifford and Sorbie, 1985; Nesyn *et al.*, 1989). In view of this complexity and the paucity of studies for curved tubes, Gallego and Shah (2009) and Ahmed Kamel (2011) concluded that the origins of their results are indeterminate and thus require further investigation. In contrast to the numerous studies on polymer DR additives to gas-liquid flows in straight tubes (Sylvester and Brill, 1976; Al-Sarkhi and

Soleimani, 2004), there are no related studies for coiled tubes. This presents further scope for future research in the field of two-phase flow in coiled tubes.

Investigators & Methodology	**Year**	**Flow configuration & coil geometry**	**Mean bubble size**	**Void fraction or concentration**	**Drag reduction**
			Air bubbles		
Shatat *et al.* Experimental	2009a &b	d_t=20mm D_c=800,400,200mm δ=0.025,0.05,0.1 H=40mm 1,000<Re<100,000 We<1.0 Laminar and turbulent bubbly flow	$d_{b,m}$=0.06mm $d_{b,max}$=0.174mm No deformation of bubbles.	0.21<VF<0.44 %	16% for δ=0.025. For a straight pipe 51% *DR*, ceteris paribus. DR effect starts at the critical *Re* number. DR increases with *VF* for all cases. The curvature of the coils had a negative effect on drag reduction. The *Re* number corresponding to the maximum DR was shifted to a higher value (compared to a straight tube). This shift increased with an increase in the curvature of the coil.
Saffari *et al.* Experimental	2013	d_t=12,19mm D_c=200mm δ=0.06,0.095 H=24mm P=0.101MPa 10,000<Re<50,000 Turbulent bubbly flow	$d_{b,m}$=0.27mm Bubble diameter decreased at higher *Re* numbers. At lower *Re* numbers, bubbles were less spherical in shape (less rigid). This is due to the influence of flow stress and reduced surface tension (in comparison to the smaller bubbles).	0.01<VF<0.09	DR increased with *VF* with a maximum of 25% at a *VF* of 9%. DR diminished with higher *Re* numbers. At a low *VF* of 1%, a DR of 9% was measured. DR diminished with an increase in the curvature of the coil.
Saffari and Moosavi Numerical (Eulerian-Eulerian multiphase model)	2014	d_t=16,25,40mm D_c=100,200mm δ=0.08,0.125,0.20 H=20,60 15,000<Re<80,000 Turbulent bubbly flow	$d_{b,m}$=0.1mm No deformation of bubbles.	0.01<VF<0.09	Due to a reduction in the mixture density, higher *VF* yields lower pressure drops, shear stress and friction coefficient.

Surfactant solutions & Foam fluids					
Weber *et al.* Experimental	1991	d_t=10.5,16.5mm 157<D_c<454mm 0.105<δ<0.036, N=12,18,34,39 1,500<Re<100,000 6,750<Re_{crit}<9,480 30°C<T<90°C Laminar and turbulent	Fluid was assumed to be quasi-Newtonian.	C=62.5;250;1,000 ppm Habon in water.	For laminar flow, surfactant additives increased the fluid drag. For turbulent flow the increase in DR with C was marginal. DR in curved tubes diminished at a lower Re value than that in straight tube, ceteris paribus.
		$f_{Fanning} = \frac{1855\delta^{\frac{2}{3}}}{Re} + 0.011$			
Gasljevic and Matthys Experimental	1999	d_t=2mm D_c=200mm δ=0.01 1.8<V<7m/s T=25°C Laminar and turbulent	Fluid was assumed to be quasi-Newtonian.	C=2,000ppm SPE95285 (Same viscosity as water)	DR in coiled tube is 30%, in a straight tube 60%, ceteris paribus. Calculated 70% reduction in turbulence effects for both straight and coiled tubes. At V>5m/s DR effect diminishes due to micelle degradation.
Inaba *et al.* Experimental	2000	d_t=17.7mm D_c=177,300.9,442.5,885mm δ=0.02,0.04,0.059,0.1 400<Re'<200,000 10°C<T<25°C θ=45°,90°,180°,270° Laminar and turbulent	Non-Newtonian viscoelastic fluid.	530<C<1,773 ppm Dodecyltrimethyl Ammonium Chloride ($C_{12}H_{25}N(CH_3)_3$ =263.89) and Sodium Salicylate ($C_7H_5NaO_3$=160.10) in water	No DR at laminar flow conditions, whilst DR at turbulent flow conditions was less in relation to that in a straight pipe. At a C of 561ppm no DR was measured. Due to the suppression of turbulence vortexes, the heat transfer coefficient was less than that for water.
		$f_{Darcy} = 6.75\left(\frac{D_c}{d_t}\right)^{-0.560} \theta^{0.146} De'^{-0.5}$ 0.02<δ<0.05; 45°<θ<270°; C>1,000ppm (SD=9.17%)			
Broniarz-Press *et al.* Experimental	2002	0.0219<δ<0.0792 1,200<Re_{gen}<30,000 70<De''<3,000 T=303,323,333K Laminar and turbulent	Non-Newtonian viscoelastic fluid.	WC=0.1,0.25 % Cationic Hexadecyltrimethylammonium chloride (HTAC) and	DR is only evident above a critical C. This contrasts to polymers where DR is significant with minute C of polymer additives.

				anionic soaps, sodium & potassium oleates with WC=2.5,7% sodium salicylate (NaSal), sodium chloride, and potassium chloride solution additives in water.	With polymer additives, DR is only evident when the molar mass is above a critical value. Cylindrical micelles stabilise the mechanisms of curved flow. DR increases with higher turbulence.
Broniarz-Press *et al.* Experimental	2003	$0.0219<\delta<0.0792$ $1{,}200<Re_{gen}<30{,}000$ $70<De''<3{,}000$ T=303,313,333K Laminar and turbulent	Non-Newtonian viscoelastic fluid.	WC=0.1,0.25 % Cationic Hexadecyltrimethylammonium chloride (HTAC) and anionic soaps, sodium & potassium oleates with WC=2.5,7% sodium salicylate (NaSal), sodium chloride, and potassium chloride solution additives in water.	DR observed in turbulent and pseudolaminar flows. Surfactant additives diminished the tube curvature effect on the friction factor. This was attributed to the damping of the secondary flow streams.
Inaba *et al.* Experimental	2005	d_t=14.4mm D_c=540mm δ=0.0267 H=32mm N=10 $10{,}000<Re'<100{,}000$ $100<De/De'<10{,}000$ $100<Gz/Gz'<10{,}000$ $5^{\circ}C<T<20^{\circ}C$ Laminar and turbulent	Non-Newtonian viscoelastic behaviour at high concentrations (>3,000ppm)	$1{,}000<C<3{,}5000$ppm Mixture of oleyldihydroxyethylamineoxide (ODEAO, $C_{22}H_{45}NO_3$=371) 90%, non-ionic surfactant & cetyldimethylaminoaciticacid betaine (CDMB, $C_{20}H_{41}NO_2$=327) 10% as a zwitterion surfactant in water.	43% DR in the coiled tube. 77% DR in a straight tube. This is due to the secondary flow that contributes towards the pressure drop in coiled tubes. Drop in the heat transfer coefficient with surfactant C. DR increases with surfactant C.

		$$\frac{f_{c,Darcy}}{f_{st,Darcy}} = De'^{0.42} C_{nd}^{0.11} T_{nd}^{1.5}$$ *where*: $$T_{nd} = \frac{T_{actual}}{T_{critical(275K)}}; C_{nd} = \frac{C_{actual}}{C_{critical\ (1{,}000ppm)}}$$			
Aly *et al.* Experimental	2006	d_t=14.4mm D_c=320,540,800,mm $0.018<\delta<0.045$ H=32mm N=10 1,000<Re'<100,000 5°C<T<20°C Laminar and turbulent	Newtonian fluids for C<3,000ppm.	250<C<5,000 ppm Mixture of non-ionic surfactant oleyldihydroxyethylamineoxide (ODEAO, $C_{22}H_{45}NO_3$=371) 90%, & cetyldimethylaminoaciticacid betaine (CDMB, $C_{20}H_{41}NO_2$=327) 10% as a zwitterion surfactant in water.	DR increased with surfactant C, with a max. of 59% at Re'=55,350 and C=2678ppm. DR increased with temperature and decreased with higher coil curvatures. Lower DR and losses in the heat transfer coefficient were measured when compared to straight tubes, ceteris paribus.
		$$f_{Darcy} = \frac{137\delta^{0.62}(1 + 0.94Cc^{-0.34}Tc^{-1.57})}{(1.56 + logDe')^{5.73}}$$ (SD=10%) 1<Tc<1.065; 4<Cc<14; $0.018<\delta<0.045$			
Gasljevic and Matthys Experimental	2009	d_t=12mm δ=0.043,0.067,0.116 0.9<V<7m/s T=25°C Laminar and turbulent	Non-Newtonian viscoelastic fluid.	C=2,000ppm cationic surfactant Ethoquad T-13 & 2,000ppm NaSl as a counterion.	DR for turbulent flow in the range of 30-40% was measured. This is less that that in a straight pipe where 75% DR was measured, ceteris paribus. DR decreased with higher curvature ratios. For the coil with the highest curvature, at V=0.9m/s, the pressure drop increased in relation to that of water. This was attributed to the higher viscosity of the surfactant solution in relation to water at a shear rate of 500s^{-1}.
Kamel and Shah Experimental	2013	11.0<d_t<63.5mm 360<D_c<2,850mm $0.01<\delta<0.031$ 20,000<Re<200,000 Turbulent	Non-Newtonian viscoelastic fluid.	VC=1.5,2.3,4 % Tallowalkylamidopropyl	DR is significant in coiled tubes and increases with C, with a significant

				dimethylamine oxide viscoelastic surfactant (VES) containing 50-65% *WC* active surfactant, 25-40% propylene glycol and water as solvents.	increase above a *VC* of 2%. Higher *C* also exhibit higher resistance to mechanical degradation. Surfactant based fluids are more resistant to shear degradation than polymer based fluids. Larger tube diameters and smaller curvature ratios yield larger DR.
		$f_{Fanning} = (-32{,}200.42\delta^3 + 1{,}830.62\delta^2 + 0.32)Re_{gen}^{[7{,}210.95\delta^3 - 316.97\delta - 0.55]}$ *where*: $Re_{gen} = \left(\frac{d_t^n V^{2-n}\rho}{8^{n-1}K}\right)$			
Wang *et al.* Numerical	2015	d_t=7.3mm D_c=203mm δ=0.036 V=3m/s	Compressible Non-Newtonian foam fluid.	65<Γ<98	The secondary flow effect (vortex roll) of the foam fluid is smaller than that of water.
Polymer solutions					
Barnes and Walters Experimental	1969	d_t=8,9.6mm 60<D_c<3000mm 0<Q<80cm^3/s T=20°C Spiral coil Laminar and turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	*VC*=0.025,0.03,0.05,0.10% Polyacrylamide (P250); Polyethylene oxide (Polyox SR305) and Guar Gum.	Easier to pump viscoelastic liquids in curved tubes. Suppression of turbulence with polymer additives which renders the flow almost laminar. Curvature enhances DR in the transition region, whilst it reduces DR at high *Re* numbers.
Kelkar and Mashelkar Experimental	1972	d_t=12.5mm D_c=665mm δ=0.019 H=38mm N=6 10<Re<100,000 Laminar and turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	50<*C*<500ppm Polyacrylamide (AP30&ET597) 0.76<*n*<1.00	DR limited to turbulent flow. DR increases with polymer *C* up to a critical *Re* when DR diminishes.
		$\beta = 0.2 + \frac{0.8}{1 + (N'_{De})^{0.8}}$ *where*: $N'_{De} = \frac{\left(\frac{V}{D_c}\right)Re^{0.75}}{\left(\left(\frac{V}{D_c}\right)Re^{0.75}\right)_{dr=0.6}}; \beta = \frac{0.6}{Cst}$			
Mashelkar and Devarajan Experimental	1976	d_t=12.48,12.49,12.50mm 92.3<D_c<1,282mm 0.01<δ<0.135 H=38.1mm 3<N<40	Non-Newtonian viscoelastic fluid. Solvent:	0.01<*C*<0.5% Carboxymethyl cellulose (CMC), Polyacrylamid	The PEO and PAA polymer yielded the best DR, even at the lowest *C*. This was attributed to the fluid

		$10<Re_{gen}<100{,}000$ $70<De<400$ $40<Wi<950$ Laminar and turbulent	Water	e (PAA-AP-30) $0.354<n<0.99$ Polyethylene oxide (PEO-WSR-301) $0.871<n<0.99$	elasticity.
		$f_{p,Fanning} = f_s(1 - 0.03923Wi^{0.2488})$ where: $f_{s,Fanning} = (9.069 - 9.438n + 4.374n^2)\delta^{0.5}De''^{(-0.768+0.122n)}$ $0.35<n<1$			
Oliver and Asghar Experimental	1976	$6.72<d_t<14.0$mm $0.033<\delta<0.082$ $152<L/d_t<410$ N=3-4 $60<De<2{,}000$ $10<Gz<400$ Laminar	Non-Newtonian viscoelastic fluid. Solvent: Water.	$250<C<2{,}500$ ppm Polyacrylamide Separan AP273 in water and a 56/44 (*WC*) glycerol/water solution with 500ppm Separan AP273.	Some DR due to the partial suppression of the secondary flow.
Rao Experimental	1993	d_t=9.35mm $98<D_c<247$mm $0.038<\delta<0.095$ H=19.5mm $8<N<20$ $10{,}000<Re<60{,}000$ Turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	C=50,100,200 ppm Polyacrylamide (Praestol 2273TR)	Higher DR with higher polymer C and smaller coil curvatures.
Azouz *et al.* Experimental	1998	d_t=30mm pH=9,10,11 $100<Re_{gen}<100{,}000$ Laminar and Turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	C=35,40 lb/kgal Linear Guar gum & Hydroxypropyl Guar (HPG), Crosslinked Guar gum & Hydroxypropyl Guar (HPG) with 12% sol. of boric acid as crosslinking agent.	For borate-crosslinked HPG, the pressure gradient is a strong function of pH and the tube length. For borate crosslinked guar, the pressure gradient is pH dependent but is not effected by the tube length.

Shah and Zhou Experimental	2001	d_t=25.4,38.1,60.3mm D_c=121.92,182.88,281.94 mm δ=0.0113,0.0165,0.0169 P_{max}=34.47MPa $4{,}000<Re_{gen}<200{,}000$ Laminar and Turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	Guar C=2.397 kg/m^3 $0.642<n<0.72$ C=3.595 kg/m^3 $0.527<n<0.55$ C=4.793 kg/m^3 $0.433<n<0.483$ partially hydrolysed polyacrylamide (PHPA), C= 2.397 kg/m^3 $0.355<n<0.384$ C=4.793 kg/m^3 $0.305<n<0.322$ Xathan gum C=1.198 $0.472<n<0.489$ C=2.397 $0.381<n<0.439$ C=4.793 $0.277<n<0.343$ hydroxyethylcellulose (HEC) C= 2.397 $0.6<n<0.668$ C=3.595 $0.494<n<0.545$ C=4.793 $0.42<n<0.443$	DR of polymer solutions decreases with the curvature ratio. Xathan and PHPA yielded the best DR properties. HEC resulted in no DR. Higher DR with smallest tube diameters. For the largest tube diameter, higher polymer C decreased the onset of the DR whilst the opposite effect was reported for the smallest tube diameter.
Shah *et al.* Experimental	2006	d_t=11mm D_c=35.60,57.24,109.97mm δ=0.01,0.019,0.031 N=3,6 $22{,}000<Re_s<155{,}000$ Turbulent	For $0.01<C<0.07$% fluid is assumed to be Newtonian. Non-Newtonian viscoelastic fluid for $C>0.07$%. Solvent: Water	Nalco ASP-820 (PHPA) $0.01<VC<0.15$% $0.814<n<1.00$	Optimum VC of ASP-820 is 0.07%. At 0.07%, ASP-820 yields a DR of 75% in straight tube and 65% in coiled tube, ceteris paribus. Increase in flow rate increases the DR while the opposite effect was reported for an increase in curvature. An increase in the polymer C or curvature ratio delays the onset if DR.

		$f_{p,Fanning} = A'\delta^{B'}\left(\frac{1.0}{Re_s^{C'}}\right)$ where *A*',*B*'&*C*' are constants given in Shah and Ahmed Kamel, (2005) and is valid for *VC*=0.07%. (*ME*= ±6%)			
Zhou *et al.* Experimental	2006	d_t=11.05mm D_c=12.14,29.67,47.70,91.64mm δ=0.010,0.019,0.031,0.076 *N*=3,6,7 5,000<Re_{gen}<100,000 Laminar and turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	*C*=10,20,30 lb/Mgal Guar gum, *C*=10,15,20,30 lb/Mgal Hydroxypropyl Guar (HPG), *C*=10,20,30 lb/Mgal Xanthan gum	DR in coiled tubing is diminished (by 10-30%) in relation to that in a straight tube, ceteris paribus. DR in coiled tubing is increased with higher *Re*. This contrasts to the case of straight tubes, where DR diminishes at higher *Re*. DR increased with *C* of Xanthan. Curvature delayed the onset of DR as a result of the delay in the onset of turbulence.
Gallego and Shah Experimental	2009	d_t=11,20.57mm D_c=35.60,57.24,109.97,182.88cm δ=0.01, 0.0113, 0.019,0.031 22,000<Re_s<430,000 *T*=21.1,37.7,54.4°C Turbulent	For 0.01<*C*<0.07 % fluid is assumed to be Newtonian. Non-Newtonian viscoelastic fluid for *C*>0.07%. Solvent: Water	Nalco ASP-700 & ASP-820 (PHPA) *VC*=0.05,0.07, 0.10,0.15% 0.75<*n*<1.00	DR decreases with curvature. DR in coiled tubes is lower than that in straight tubes, ceteris paribus. At 0.07% ASP-820, DR is 77% in a straight tube and 64% in the coiled tube (79%&59% for ASP-700). The increase in *T* resulted in a decrease of DR in straight tubes. The opposite effect was measured in coiled tubes (DR=45%,52%&55% at 21.1,37.7,54.4°C respectively for ASP-820) DR decreases with tube roughness in both straight and coiled tubes (64% to 60% for coiled tube).
		$N_{De} = \left[\frac{1.6675 * 10^{-3}\left(f_{s,Fanning}Re_s\right)^{1.4084}\left(\frac{8\lambda\upsilon}{d_t}\right)}{\left(1 + 1.0974 * 10^{-3}\left(f_{s,Fanning}Re_s\frac{8\lambda\upsilon}{d_t}\right)^{1.42305}\right)^{0.7511}}\right]\left[\frac{\rho_p\mu_s}{\rho_p\mu_o}\right]^{0.1129}$			

		$N_{De}=\left(\frac{f_{s,Fanning}}{f_{p,Fanning}}\right)^2-1$ (ME= ±10%)			
Shah and Zhou Experimental	2009	d_t=12mm D_c=146,356,572, 1100 mm δ=0.01,0.019,0.031,0.076 N=3,3,7 3,700<Re_{gen}<11,500 Laminar and turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	1.198<C<3.595 kg/m^3 Guar gum, 0.482<n<0.819 Hydroxypropyl Guar (HPG), 0.485<n<0.805 Xanthan gum 0.310<n<0.717	Significant DR with all three polymer fluids. Curvature reduces the DR and delays the onset of DR.
		$\frac{1}{\sqrt{f_{Fanning}}}=\frac{1}{0.05311+0.29465\delta^{0.5}}log_{10}\left(Re_{gen}f^{\frac{1}{2}}\right)+\frac{1}{0.03094+0.24575\delta^{0.5}}$			
Ogugbue and Shah Numerical	2011	δ=0.3,0.5,0.6,0.8 ε=0,0.25,0.5,0.75,0.96 100<Re_{gen}<10,000 Laminar and turbulent	Non-Newtonian viscoelastic fluid. Solvent: Water	C=20,30,40,60 lb/Mgal Guar 0.335<n<0.666	DR increases with increased eccentricity (50% reduction for fully eccentric annular section) Higher C increased the frictional pressure drop for laminar flow. For turbulent flow, all C resulted in a significant DR.
		$f_{Fanning}=0.00378\frac{d_{eit}}{d_{iot}}+\frac{3.7374}{Re_{gen}}+\frac{4042}{2Re_{gen}}-0.00124$ (ME= ±5%)			
Ahmed Kamel Experimental	2011	d_t=11mm D_c=579mm δ=0.019 T=22,35,38°C 20,000<Re<200,000 P_{max}=6.9MPa Turbulent	Properties assumed to be quasi-Newtonian. Solvent: Water	Nalco ASP-820 (PHPA) VC=0.07% $n\approx1.00$	DR in the range of 30-80% At elevated T, the DR effect is diminished in straight tubes while it remains quasi- constant in coiled tubes.
		$\frac{DR_T}{DR_a}=1.0$ (ME= ±2.1%)			

Table 1: Review of the experimental and numerical work

5. Conclusions

The studies reviewed have demonstrated that, due to the secondary flow, which increases with curvature, DR in coiled tubes is diminished when compared to straight tubes. However, a significant DR can be still be achieved with the introduction of: bubbles (9-25%), surfactant (30-59%) and polymer (circa 30-80%) additives. DR is a strong function of the surfactant concentration and the air volume fraction whilst with polymer additives DR efficiency is dependent on the molecular weight, structure and solubility. DR is generally present in flows with *Re* numbers in excess of the critical number. However, at elevated *Re* numbers DR diminishes. This is due to the higher centrifugal forces (air bubbles and polymers) and mechanical degradation with high shear stress (surfactants). A number of authors have presented correlations for the calculation of the friction factor which are typically a function of the: curvature ratio, *Re* and *De* numbers and the additive concentration.

Due to their low molecular weights, viscous properties and resilience to mechanical degradation, surfactant based fluids are generally considered to be superior to polymer based fluids. Hence, surfactants are suitable for a variety of applications such as district cooling and heating systems. A significant scope for future research has been elucidated for DR in coiled tubes with the injection of air bubbles (impact of bubble size and relation with the Lockhart and Martinelli correlation) and the application of a combination of methods, such as the use of polymer and surfactant additives with bubbly flow.

Acknowledgments

The authors of the current investigation would like to thank the University of Central Lancashire UK, for facilitating the completion of this study as well as the various authors who have been contacted during the course of this study.

Notation List

C	concentration (ppm)
Cc	non-dimensional surfactant concentration (-)
Cst	empirical constant (-)
d	tube internal diameter (m)
dr	drag ratio (-)
D	helix diameter (m)
De	Dean number ($Re\delta^{1/2}$) (-)
De'	modified Dean number ($Re'\delta^{1/2}$) (-)
De''	modified Dean number ($Re_{gen}\delta^{1/2}$) (-)
DR	drag reduction (%)
f	friction factor (-)
FC	friction coefficient (-)
Gz	Graetz number ($RePr/z$) (-)
Gz'	modified Graetz number ($Re'Pr'/z$) (-)
H	pitch (m)
K	rheometric and technical consistency index (Pa s^n)
L	length (m)
ME	mean error (%)
n	power law model flow behaviour index (-)
N	number of turns (-)
N_{De}	Deborah number (-)
N_{De}'	modified Deborah number (-)

P	pressure (Pa)
Pr	Prandtl number (-)
Pr'	modified Prandtl number (-)
Q	volume flow rate (m^3/s)
Re	Reynolds number (-)
Re'	modified Reynolds number as proposed by Metzner and Reed (1955) $\left[8^{1-n}\left(\frac{3n+1}{4n}\right)\left(\frac{V^{2-n}d_t^n\rho}{K}\right)\right]$ (-)
Re_{crit}	critical Reynolds number $(2E4\delta^{0.32})$ (-)
Re_{gen}	generalised Reynolds number $\left(\frac{V^{2-n}d_t^n\rho}{8^{n-1}K}\right)$ (-)
SD	standard deviation (%)
T	temperature (^{0}C)
Tc	non-dimensional surfactant solution temperature (-)
TRD	turbulence reduction: drag (-)
V	flow velocity (m/s)
VC	volume concentration (%)
VF	volumetric void fraction (-)
We	Weber number (-)
Wi	Weissenberg number (σ_{el}/σ_v) (-)
WC	weight concentration (%)
x	axial distance of coiled pipe (m)
z	dimensionless axial distance (x/d_t) (-)

Greek

β	reduced friction factor (-)
δ	curvature ratio (-)
ε	coil eccentricity (-)
θ	angle from inlet of curved pipe (^{o})
λ	relaxation time (s)
μ	viscosity (cP)
μ_o	zero shear rate viscosity (cP)
v	average fluid velocity (ft/s)
ρ	density (kg/m^3)
σ	stress (N/m^2)
Γ	quality (%)

Subscripts

a	ambient temp
b	bubble
bf	base fluid
c	coil
crit	critical
DRF	drag reducing fluid
eff	effective
el	elastic
eit	external diameter of inner tubing
gen	generalised
iot	internal diameter of outer tubing

l	liquid
lm	laminar
m	mean
nd	non-dimensional
o	zero
p	polymer solution
s	solvent
st	straight tube
t	tube
tb	turbulent
tp	two-phase
T	elevated temperature
v	viscous

References

Ahmed Kamel A.H., 2011, Drag reduction behaviour of polymers in straight and coiled tubing at elevated temperature, Oil and Gas Business,1, pp. 107-128.

Akagawa K., Tadashi S., Minoru U., 1971, Study on a gas-liquid two-phase flow in helically coiled tubes, Bulletin of JSME, 14 (72), pp. 564-571

Al-Sarkhi A., Hanratty T.J., 2001, Effect of drag-reducing polymer on annular gas-liquid flow in horizontal pipe, International Journal of Multiphase Flow, 27(7), pp. 1151-1162.

Al-Sarkhi A., Soleimani A., 2004, Effect of drag reducing polymers on two-phase gas-liquid flows in a horizontal pipe, Transactions of the Institution of Chemical Engineers, Chemical Engineering Research and Design 82(A12), pp.1583-1588.

Al-Sarkhi A., 2010, Drag reduction with polymers in gas/liquid-liquid flows in pipes: A literature review, Journal of Natural Gas Science and Engineering, 2, pp. 41-48.

Aly W.I., Inaba H., Haruki N., Horibe A., 2006, Drag and heat transfer reduction phenomena of drag-reducing surfactant solutions in straight and helical pipes, Special Issue on Boiling and Interfacial Phenomena: Forced Convection, American Society of Mechanical Engineers, 128(8) pp. 800-810.

Aly W., 2014, Numerical study on turbulent heat transfer and pressure drop of nanofluid in coiled tube-in-tube heat exchangers, Energy Conversion and Management, 79, pp. 304-316.

Azouz I., Shah S.N., Vinod P.S., Lord D.L.,1998, Experimental investigation of frictional pressure losses in coiled tubing, Society of Petroleum Engineers: Production and Facilities, 13(2), Society of Petroleum Engineers-37328-PA.

Barnes H.A., Walters K., 1969, On the flow of viscous and elastico-viscous liquids through straight and curved pipes, Proceedings of the Royal Society, London, A314(1516), pp. 85-109.

Bird R.B., Armstrong R.C., Hassager O., 1987, Dynamics of polymeric liquids, Second Edition, John Wiley & Sons, New York, Vol. I.

Broniarz-Press L., Rozanski J., Dryjer S., Woziwodzki S., 2002, Flow of the surfactant's solutions in both straight and curved pipes, Proceedings of the 15th International Congress of Chemical Process Engineering, CHISA'2002, Praha.

Broniarz-Press L., Rozanski J., Dryjer S., Woziwodzki S., 2003, Characteristics of the flow of the surfactants solutions in curved pipes, International Journal of Applied Mechanical Engineering, 8, Special Issue: ICER'2003 pp. 135-139.

Broniarz-Press L., Rozanski J., Rozanska S., 2007, Drag reduction effect in pipe systems and liquid falling film flow, Reviews in Chemical Engineering, 23(3-4), pp.149-245.

Clifford P.J., Sorbie K.S., 1985, The effects of chemical degradation on polymer flooding, The International Symposium on Oilfield and Geothermal Chemistry, Phoenix, Arizona, 9-11th April.

Fsadni A.M., Whitty P.M., 2016, A review on the two-phase pressure drop characteristics in helically coiled tubes, Applied Thermal Engineering, 103, pp. 616-638.

Fujiwara A., Minato D., Hishida K., 2004, Effect of bubble diameter on modification of turbulence in an upward pipe flow, International Journal of Heat and Mass Transfer, 25(3), pp. 481-488.

Gallego F., Shah S.N., 2009, Friction pressure correlations for turbulent flow of drag reducing polymer solutions in straight and coiled tubing, Journal of Petroleum Science and Engineering, 65, pp. 147-161.

Gasljevic K., Matthys E.F., 1997, Experimental investigation of thermal and hydrodynamic development regions for drag-reducing surfactant solutions, Journal of Heat Transfer, 119(1), pp. 80-88.

Gasljevic K., Matthys E.F, 1999, Improved quantification of the drag reduction phenomenon through turbulence reduction parameters, Journal of Non-Newtonian Fluid Mechanics, 84(2-3), pp. 123-130.

Gasljevic K., Matthys E.F., 2009, Friction and heat transfer in drag-reducing surfactant solution flow through curved pipes and elbows, European Journal of Mechanics B/Fluids, 28, pp. 641-650.

Inaba H., Haruki N., Horibe A., 2000, Flow and heat transfer characteristics of water solution with drag reduction additive in curved tubes, Proceedings of the Symposium on Energy Engineering in the 21st century, SEE 12(J6) pp. 723-730.

Inaba H., Aly W.I.A., Haruki N., Horibe A., 2005, Flow and heat transfer characteristics of drag reducing surfactant solution in a helically coiled pipe, Heat Mass Transfer, 41, (10), pp. 940-952.

Ito H., 1959, Friction factors for turbulent flow in curved pipes, ASME Journal of Basic Engineering, 81, pp. 123-134.

Kamel A.H., Shah S.S., 2013, Maximum Drag Reduction Asymptote for Surfactant-Based Fluids in Circular Coiled Tubing, Journal of Fluids Engineering, 135 (3) 031201-1:10.

Kelkar J.V., Mashelkar R.A., 1972, Drag reduction in dilute polymer solutions, Journal of Applied Polymer Sciences, 16, pp. 3047-3062.

Kostic M., 1994, On turbulent drag and heat transfer phenomena reduction and laminar heat transfer enhancement in non-circular duct flow of certain non-Newtonian fluids, International Journal of Heat and Mass Transfer, 37 (Suppl. 1) pp. 133-147.

Liu T.J., 1993, Bubble size and entrance length effects in void development in a vertcial channel, International Journal of Multiphase Flow, 19(1), pp.99-113.

Mashelkar R.A., Devarajan G.V., 1976, Secondary flow of non-Newtonian fluids. II. Frictional losses in laminar flow of purely viscous and viscoelastic fluids through coiled tubes, Transactions of the Institution of Chemical Engineers, 54, pp. 108-114.

McCormick M.E., Bhattacharya R., 1973, Drag reduction of a submersible hull by electrolysis, Naval Engineering Journal, 85(2), pp. 11-16.

Merkle C.L., Deutsch S., 1992, Microbubble drag reduction in liquid turbulent boundary layers, Applied Mechanics Reviews, 45(3), pp. 103-127.

Metzner A.B., Reed J.C., 1955, Flow of non-Newtonian fluids – correlation of the laminar, transition and turbulent flow regions, American Institute of Chemical Engineers Journal, 1(4) pp. 434-440.

Mohammed H.A., Narrein K., 2012, Thermal and hydraulic characteristics of nanofluid flow in a helically coiled tube heat exchanger, International Communications in heat and Mass Transfer, 39, pp. 1375-1383.

Moriguchi Y., Kato H., 2002, Influence of microbubble diameter and distribution on frictional resistance reduction, Journal of Marine Science and Technology, 7, pp. 79-85.

Murai Y., Fukuda H., Oishi Y., Kodama Y., Yamamoto F., 2007, Skin friction reduction by large air bubbles in a horizontal channel flow, International Journal of Multiphase Flow, 33(2) pp. 147-163.

Murai Y., 2014, Frictional drag reduction by bubble injection, Experiments in Fluids, 55:1773 pp. 1-28.

Nesyn G.V., Manzhai V.N., Shibayev V.P., 1989, Influence of temperature and the nature of the solvent on the ability of polymers to lower the drag resistance of liquids, Polymer Science U.S.S.R., 31(7) pp. 1546-1553.

Nouri N.M., Motlagh S.Y., Navidbakhsh M., Dalilhaghi M., Moltani A.A., 2013, Bubble effect on pressure drop reduction in upward pipe flow, Experimental Thermal and Fluid Science, 44, pp. 592-598.

Ogugbue C.C., Shah S.N., 2011, Laminar and turbulent friction factors for annular flow of drag-reducing polymer solutions in coiled-tubing operations, Society of Petroleum Engineers: Drilling and Completion, 26(4) pp.506-518.

Oliver D.R., Asghar S.M., 1976, Heat transfer to Newtonian and viscoelastic liquids during laminar flow in helical coils, Transactions of the Institution of Chemical Engineers, 54, pp. 218-224.

Rao B.K., 1993, Turbulent heat transfer to viscoelastic fluids in helical passages, Experimental Heat Transfer, 6(2), pp. 189-203.

Saffari H., Moosavi R., Gholami E., Nouri N.M., 2013, The effect of bubble in the pressure drop reduction in helical coil, Experimental Thermal and Fluid Science, 51, pp. 251-256.

Saffari H., Moosavi R., 2014, Numerical study of the influence of geometrical characteristics of a vertical helical coil on a bubbly flow, Journal of Applied Mechanics and Technical Physics, 55(6), pp.957-969.

Shah S.N., Zhou Y., 2001, An experimental study of drag reduction of polymer solutions in coiled tubing, The University of Oklahoma, Coiled Tubing Consortium Annual Meeting, Houston, Texas, Society of Petroleum Engineers: 68419.

Shah S.N., Ahmed Kamel A.H., 2005, Drag reduction in straight and coiled tubing, The University of Oklahoma, Coiled Tubing Consortium Annual Meeting, Houston, Texas.

Shah S.N., Kamel A., Zhou Y., 2006, Drag reduction in straight and coiled tubing – An experimental study, Journal of Petroleum and Engineering, 53, pp. 179-188.

Shah S.N., Zhou Y., 2009, Maximum drag reduction asymptote of polymeric fluid flow in coiled tubing, American Society of Mechanical Engineers, Journal of Fluids Engineering, 131(1) pp. 011201-1:9.

Shatat M.M.E., Yanase S., Takami T., Hyakutake T., 2009a, Drag reduction effects of micro-bubbles in straight and helical pipes, Journal of Fluid Science and Technology, 4(1), pp.156-167.

Shatat M.M.E., Yanase S., Takami T., Hyakutake T., 2009b, Pressure drop characteristics of water flow with micro-bubbles through helical pipes, Proceedings of the 6th WSEAS International Conference on Fluid Mechanics, ISSN: 1790-5095.

Shen X., Ceccio S.L., Perlin M., 2006, Influence of bubble size on micro-bubble drag reduction, Experiments in Fluids, 41(3), pp. 415-424.

Sylvester N.D., Brill J.P., 1976, Drag-reduction in two-phase annular mist flow of air and water, American Institute of Chemical Engineers Journal, 22(3), pp.615-617.

Toms B.A., 1948, Some observations on the flow of linear polymer solutions through straight tubes at large Reynolds Numbers, Proceedings of the First International Congress on Rheology, North Holland, Amsterdam.

Wang F., Li Z., Chen H., Li S., 2015, Foam fluid flow analysis in helical coiled tubing using CFD, Procedia Engineering, 126, pp. 696-700.

Weber M., Steiff A.S., Weinspach P.M., 1991, Heat transfer and pressure loss in flow of surfactant solutions in coiled pipes, Forschung im Ingenieurwesen, 57(4), pp. 112-118 (In German).

Wei T., Willmarth W.W., 1992, Modifying turbulent structure with drag-reducing polymer additives in turbulent channel flows, Journal of Fluid Mechanics, 245, pp. 619-641.

Zhou Y., Shah S.N., Gujar P.V., 2006, Effect of coiled tubing curvature on drag reduction of polymeric fluids, Society of Petroleum Engineers: Production and Operations, 21(1), pp.134-141.

Highlights

- Review on pressure drop reduction studies in helically coiled tubes
- Air bubbles, surfactant and polymer additives are effective in diminishing drag
- Drag reduction is diminished in relation to straight tubes
- Drag reduction is predominantly evident in turbulent flow
- Drag reduction diminishes with higher coil curvatures and excessive *Re* numbers